

Superheated Steam At 500 Kpa And 300 Degrees C Expanding Isentropically To 50 Kpa What Is Final State

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Understanding the thermodynamic behavior of superheated steam is essential in numerous industrial applications, including power generation, heating, and propulsion systems. When superheated steam expands isentropically—meaning at constant entropy—it undergoes a change in pressure and temperature, leading to a new state that can be precisely determined using thermodynamic properties and steam tables. This article explores the process where superheated steam at 500 kPa and 300°C expands adiabatically and reversibly to a pressure of 50 kPa, and it aims to determine the final state parameters such as temperature, specific volume, and entropy.

Introduction to Superheated Steam and Isentropic Expansion

Superheated steam is steam that exists at a temperature higher than its saturation temperature at a given pressure. Unlike saturated steam, which contains a mixture of liquid and vapor, superheated steam is entirely vapor and possesses distinct thermodynamic properties. Its applications span turbines, heating systems, and chemical processes due to its high energy content and predictable behavior.

Isentropic expansion refers to an idealized process where the entropy remains constant throughout the expansion. This process assumes no heat transfer or frictional losses, making it an ideal model for analyzing turbines and nozzles in thermodynamics. When superheated steam expands isentropically from a high-pressure state to a lower pressure, its temperature and specific volume change according to the properties of the steam, following the principles of the first and second laws of thermodynamics.

Thermodynamic Data for Superheated Steam at 500

kPa and 300°C

To analyze the expansion process, it's essential first to determine the initial thermodynamic properties of the superheated steam at the starting conditions. According to standard steam tables:

- Initial Pressure (P_1): 500 kPa
- Initial Temperature (T_1): 300°C

From the steam tables for superheated steam at 500 kPa:

Property	Value
Specific Volume (v_1)	approximately 0.25795 m ³ /kg
Specific Entropy (s_1)	approximately 6.773 kJ/(kg·K)

Note: Values may vary slightly depending on the specific steam table used, but these are typical for superheated steam at the specified conditions.

Understanding Isentropic Expansion to 50 kPa

The process involves the following key points:

- Initial State: Superheated steam at 500 kPa and 300°C
- Final Pressure: 50 kPa
- Process Type: Isentropic (constant entropy)

Since the expansion is isentropic, the entropy at the final state (s_2) equals the initial entropy (s_1). To find the final state parameters:

1. Use the entropy value ($s_2 = s_1$) and the final pressure (50 kPa) in the superheated steam tables to determine the final temperature (T_2) and specific volume (v_2).
2. Confirm the state is superheated or saturated at the final pressure based on the entropy value.

Determining the Final State of the Steam

Step 1: Find the Final Temperature (T2)

Using the entropy value $s_2 = 6.773 \text{ kJ}/(\text{kg}\cdot\text{K})$ and the pressure $P_2 = 50 \text{ kPa}$, consult the superheated steam tables:

- At 50 kPa, the saturation temperature is approximately 81.3°C .
- The superheated steam tables list properties for various temperatures, with entropy increasing with temperature at constant pressure.

By interpolating or referencing the tables, locate the temperature at which the specific entropy matches $6.773 \text{ kJ}/(\text{kg}\cdot\text{K})$:

- At 50 kPa:
- At 150°C , $s \approx 6.318 \text{ kJ}/(\text{kg}\cdot\text{K})$
- At 200°C , $s \approx 6.624 \text{ kJ}/(\text{kg}\cdot\text{K})$
- At 220°C , $s \approx 6.767 \text{ kJ}/(\text{kg}\cdot\text{K})$
- At 230°C , $s \approx 6.823 \text{ kJ}/(\text{kg}\cdot\text{K})$

Since $s \approx 6.773$, the temperature T_2 is approximately 220°C .

Step 2: Find the Specific Volume (v2)

At 50 kPa and $T_2 \approx 220^\circ\text{C}$, the specific volume v_2 can be obtained from the superheated steam tables:

Pressure (kPa)	Temperature ($^\circ\text{C}$)	Specific Volume (m^3/kg)
50	220	approximately $0.257 \text{ m}^3/\text{kg}$

This indicates that the specific volume remains approximately the same as initial, confirming the idealized nature of the process.

Step 3: Final State Summary

Property	Value
Final Pressure (P_2)	50 kPa
Final Temperature (T_2)	approximately 220°C
Specific Volume (v_2)	approximately $0.257 \text{ m}^3/\text{kg}$
Specific Entropy (s_2)	$6.773 \text{ kJ}/(\text{kg}\cdot\text{K})$

The process results in a significant drop in temperature from 300°C to

approximately 220°C, with the entropy remaining constant, characteristic of an ideal isentropic expansion.

Additional Thermodynamic Considerations

- **Work Done by the Steam:** The work output during expansion can be estimated from the pressure-volume relationship, but specific calculations require more data.
- **Quality of the Steam:** Since the process remains in the superheated region throughout, there is no condensation, and the steam remains vapor.
- **Real-World Deviations:** Actual turbines and nozzles involve irreversibilities, which lead to entropy generation and deviations from idealized behavior.

Practical Applications of Isentropic Expansion Analysis

Understanding the final state of superheated steam after isentropic expansion allows engineers to:

- Design efficient turbines and nozzles.
- Calculate work output and thermal efficiency.
- Optimize power plant cycles such as Rankine cycles.
- Assess the energy content and suitability of steam for various processes.

Conclusion

In summary, superheated steam at 500 kPa and 300°C, when expanded isentropically to 50 kPa, reaches a final state characterized by:

- Pressure: 50 kPa
- Temperature: approximately 220°C
- Specific Volume: about 0.257 m³/kg
- Entropy: remains at 6.773 kJ/(kg·K)

This analysis demonstrates the fundamental principles of thermodynamics and the importance of steam tables in predicting the behavior of steam during expansion processes. Such knowledge is vital for designing efficient thermal

systems and understanding energy transfer mechanisms in steam-based applications.

Keywords: superheated steam, isentropic expansion, thermodynamics, steam tables, final state, pressure, temperature, specific volume, entropy, power plants, turbines

Frequently Asked Questions

What is the initial pressure and temperature of the superheated steam in the problem?

The initial pressure is 500 kPa and the initial temperature is 300°C.

What does it mean for the steam to expand isentropically in this context?

An isentropic expansion means the process occurs at constant entropy, with no entropy change, typically implying a reversible adiabatic process.

What is the final pressure of the steam after expansion?

The final pressure of the steam after expansion is 50 kPa.

How can we determine the final temperature and state of the steam after expansion?

Since the process is isentropic, we use steam tables or Mollier diagrams to find the final temperature and quality at 50 kPa, based on the initial entropy.

What is the significance of knowing the initial entropy in this process?

The initial entropy helps us track the entropy during the isentropic expansion, allowing us to determine the final state parameters accurately.

Can we assume the steam remains superheated after expansion to 50 kPa?

It depends on the final temperature; if the temperature is above the saturation temperature at 50 kPa, the steam remains superheated; otherwise,

it becomes wet or saturated.

What tools or charts are useful for analyzing this isentropic expansion?

Steam tables, Mollier h-s diagrams, or thermodynamic software are useful for determining the final state after expansion.

Why is it important to consider the process as adiabatic and reversible in this scenario?

Because the process is assumed to be isentropic, which implies no heat transfer or irreversibilities, simplifying the calculation of the final state.

What is the typical use of such an analysis in engineering applications?

This analysis helps in designing turbines, nozzles, and other expansion devices by predicting the final state and energy transfer in thermodynamic cycles.

Additional Resources

Superheated Steam At 500 Kpa And 300 Degrees C Expanding Isentropically To 50 Kpa: What Is Final State?

Introduction

The thermodynamic behavior of superheated steam under varying pressure and temperature conditions is a fundamental concern in engineering disciplines such as power plant design, process engineering, and HVAC systems. Understanding the state of steam after expansion, especially when it occurs isentropically (i.e., at constant entropy), is crucial for predicting performance, efficiency, and safety of thermodynamic cycles. This article delves deeply into the problem of determining the final state of superheated steam initially at 500 kPa and 300°C when it expands isentropically to 50 kPa.

Background and Relevance

Superheated steam is steam heated beyond its saturation temperature at a given pressure, thus existing in a gaseous phase without liquid droplets. Its thermodynamic properties are well tabulated in steam tables, which provide essential data for calculations. When superheated steam undergoes isentropic expansion—common in turbines and nozzles—the process involves a reduction in pressure accompanied by a corresponding change in temperature and specific

volume, but with entropy remaining constant.

Understanding the final state after such an expansion is vital for:

- Designing turbines and nozzles for optimal efficiency.
- Estimating work output and heat transfer characteristics.
- Ensuring safety margins in high-pressure systems.
- Conducting cycle analysis in Rankine and other thermodynamic cycles.

This investigation aims to methodically determine the final temperature, specific volume, and other relevant properties of the steam after expansion to 50 kPa.

Methodology Overview

The primary approach involves:

1. Initial State Data Collection: Using superheated steam tables at 500 kPa and 300°C.
2. Properties at Initial State: Determining specific entropy, specific volume, enthalpy, etc.
3. Isentropic Expansion Assumption: Applying the principle that entropy remains constant during the process.
4. Final State Calculation: Using the entropy value at the initial state to locate the state at 50 kPa in superheated or wet region tables.
5. Determining Final Properties: Including temperature, specific volume, enthalpy, and phase conditions.

This systematic process ensures accurate and comprehensive results.

Initial State Properties

Referring to the superheated steam tables at a pressure of 500 kPa:

Property	Value at 500 kPa and 300°C
Specific Enthalpy (h_1)	approximately 2778.5 kJ/kg
Specific Entropy (s_1)	approximately 6.718 kJ/kg·K
Specific Volume (v_1)	approximately 0.2579 m³/kg

(Values are approximate and derived from standard superheated steam tables; actual values may vary slightly depending on the data source.)

Key observations:

- The steam is clearly superheated at the initial state.
- The entropy and enthalpy are known, providing a basis for the isentropic process.

Applying the Isentropic Expansion

Since the process is isentropic:

- $s_2 = s_1 = 6.718 \text{ kJ/kg}\cdot\text{K}$

The final pressure:

- $P_2 = 50 \text{ kPa}$

The challenge is to locate the final state corresponding to s_2 at P_2 , which involves:

- Consulting the superheated steam tables at $P = 50 \text{ kPa}$.
- Checking the temperature range where $s \approx 6.718 \text{ kJ/kg}\cdot\text{K}$.

Superheated steam tables at 50 kPa:

Temperature ($^{\circ}\text{C}$)	Enthalpy (h)	Entropy (s)	Specific Volume (v)
100 $^{\circ}\text{C}$	$h \approx 2675.7 \text{ kJ/kg}$	$s \approx 7.357 \text{ kJ/kg}\cdot\text{K}$	$0.4623 \text{ m}^3/\text{kg}$
150 $^{\circ}\text{C}$	$h \approx 2720.3 \text{ kJ/kg}$	$s \approx 7.795 \text{ kJ/kg}\cdot\text{K}$	$0.4978 \text{ m}^3/\text{kg}$
200 $^{\circ}\text{C}$	$h \approx 2764.7 \text{ kJ/kg}$	$s \approx 8.211 \text{ kJ/kg}\cdot\text{K}$	$0.5310 \text{ m}^3/\text{kg}$
250 $^{\circ}\text{C}$	$h \approx 2805.3 \text{ kJ/kg}$	$s \approx 8.604 \text{ kJ/kg}\cdot\text{K}$	$0.5588 \text{ m}^3/\text{kg}$

From these data points, the entropy at 50 kPa exceeds the initial entropy ($6.718 \text{ kJ/kg}\cdot\text{K}$), indicating the final state is within the superheated region at temperatures below 100°C . Since $s = 6.718 \text{ kJ/kg}\cdot\text{K}$ is less than the values at the listed temperatures, the final state must be at a temperature below 100°C .

Interpolating to find the temperature corresponding to $s = 6.718 \text{ kJ/kg}\cdot\text{K}$:

- The entropy at 100°C is approximately 7.357, which is higher than 6.718.
- At lower temperatures, entropy decreases; thus, the final temperature should be less than 100°C .

Using linear interpolation between the saturated vapor line and the superheated data:

- At 0°C (saturated vapor), the entropy is about $7.359 \text{ kJ/kg}\cdot\text{K}$.
- At 50°C , it is approximately $7.0 \text{ kJ/kg}\cdot\text{K}$.

Since $6.718 \text{ kJ/kg}\cdot\text{K}$ is less than these values, the final state likely falls within the compressed liquid or wet region, but because the initial entropy is relatively high, the steam remains superheated at the final pressure but at a lower temperature.

Alternatively, to accurately determine the final state, it is best to consult comprehensive steam tables or Mollier diagrams for $s = 6.718 \text{ kJ/kg}\cdot\text{K}$ at $P = 50 \text{ kPa}$.

Final State Determination

Based on typical steam table data:

- The entropy of saturated vapor at 50 kPa is approximately 8.15 kJ/kg·K.
- The entropy of saturated liquid at 50 kPa is approximately 1.30 kJ/kg·K.
- The initial entropy (6.718 kJ/kg·K) is significantly less than saturated vapor entropy, indicating the final state is in the superheated region.

Interpolating within superheated tables suggests:

- At 50 kPa, the temperature corresponding to $s \approx 6.718$ kJ/kg·K is approximately 70°C.

Corresponding properties at 50 kPa and 70°C:

Property	Approximate Value
Enthalpy (h_2)	approximately 2740 kJ/kg
Specific Volume (v_2)	approximately 0.30 m³/kg

Key Conclusions:

- The final temperature after isentropic expansion to 50 kPa is approximately 70°C.
- The specific enthalpy decreases from about 2778.5 kJ/kg to approximately 2740 kJ/kg.
- The specific volume increases from 0.2579 m³/kg to roughly 0.30 m³/kg.
- The steam remains superheated at the final state, although close to saturation conditions.

Implications and Engineering Significance

This analysis underscores several important aspects:

1. Energy and Work Potential: The decrease in specific enthalpy indicates the amount of work that could be extracted during expansion.
2. Efficiency Considerations: Isentropic expansion idealizes maximum work output; real processes involve irreversibilities.
3. Design Parameters: Engineers can use these data to optimize turbine blade design, nozzle shapes, and cycle efficiencies.
4. Safety Margins: Understanding the final state conditions helps prevent equipment damage due to unexpected phase changes or temperature excursions.

Limitations and Further Considerations

While the above analysis provides a clear approximation, real-world processes may involve:

- Non-idealities and irreversibilities.
- Heat transfer with surroundings.
- Variations in steam quality or impurities.
- Deviations from the ideal isentropic assumption.

Advanced modeling might involve using software such as EES (Engineering Equation Solver) or thermodynamic simulation tools for more precise outcomes.

Conclusion

The isentropic expansion of superheated steam from 500 kPa and 300°C to 50 kPa results in a final state characterized by:

- A temperature of approximately 70°C.
- A specific enthalpy of about 2740 kJ/kg.
- A specific volume of roughly 0.30 m³/kg.
- The steam remaining superheated at this lower pressure.

This detailed analysis demonstrates how fundamental thermodynamic principles, combined with standard steam tables, enable engineers and researchers to predict system behavior accurately. Such insights are invaluable for designing efficient, safe, and reliable thermal systems in power generation and industrial applications.

References:

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Note: For precise calculations, always consult the latest and most detailed steam tables or thermodynamic software suited to your specific application.

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